

P10435 DE, x



1 375 564

(21) Application No. 8004/74 (22) Filed 27 March 1971

(23) Complete Specification filed 10 March 1972

(44) Complete Specification published 27 Nov. 1974

(51) International Classification F16D 55/224

(52) Index at acceptance

F2E 2N1A1 2N1A4A2 2N1A5 2N1D16 2N1D2B

2N1D6A 2N1D6B 2N1D6C1 2N1D6C2 D18

(72) Inventor HELMUT HEIBEL

(54) IMPROVEMENTS IN DISC BRAKES FOR VEHICLES

(71) We, GIRLING LIMITED, a British Company of Kings Road, Tyseley, Birmingham, 11, do hereby declare the invention, for which we pray that a patent may be granted to us, and the method by which it is to be performed, to be particularly described in and by the following statement:—

This invention relates to disc brakes for vehicles, the disc brakes being of the kind in which friction pad assemblies for engagement with opposite faces of a rotatable disc are located in a caliper member which straddles a portion of the peripheral edge of the disc and which is guided for movement with respect to the disc on a stationary drag-taking member adjacent the disc, and at least one of the friction pad assemblies is adapted to be applied directly to one face of the disc by actuating means, resilient means being provided for urging complementary surfaces of the members into engagement the resilient means acting between the members through a lever which engages with a part of one member and is pivotally connected to the other member.

According to our invention in a disc brake of the kind set forth for vehicles, the resilient means acts on the lever at one of a pair of relatively spaced locations, at one of which one pair or set of complementary surfaces of the members are urged into engagement, and at the other of which another different pair or set of complementary surfaces of the members are urged into engagement.

Conveniently the locations are disposed in positions such that, when the resilient means act on one of the locations, complementary circumferentially arranged surfaces on the members are urged into engagement and, when the resilient means act of the other location, complementary radially arranged surfaces on the members are urged into engagement.

Preferably the lever is formed with a pair of abutment surfaces disposed at right angles to each other and intersecting at their

inner ends for engagement with complementary surfaces on the said one member. The pivotal connection between the lever and the said other member, the abutment surfaces, and the complementary surfaces are relatively arranged so that the abutment surfaces act as a stop to prevent substantial relative movement between the members at least in a radial direction, upon failure of the resilient means.

In one construction of brake a lever and resilient means are located at each of the circumferentially spaced opposite ends of a brake and each resilient means comprises a tension spring acting between a fixed anchorage on the said one member and one of the locations on the corresponding lever which is connected to the other member by means of the pivotal connection. Thus the levers form circumferentially spaced stops, and one of the abutment surfaces on each lever acts on a complementary surface of the said one member under the influence of a tension spring. The levers may be biased by the tension springs in directions such that one lever subjects the said one member to a circumferentially directed force and the other lever subjects the said one member to a radially directed force.

One embodiment of our invention is illustrated in the accompanying drawings in which:—

Figure 1 is a transverse section through an hydraulically operated disc brake;

Figure 2 is a longitudinal half-section of the brake actuator and the stationary drag-taking member;

Figure 3 is a view on an enlarged scale of the pivotal connection for the lever shown in Figure 2; and

Figure 4 is a perspective view of a lever for incorporation in the disc brake.

In the hydraulically actuated disc brake illustrated in Figures 1 to 4 of the drawings, 1 is a stationary drag-taking member adapted to be secured to a fixed part adjacent to one face of a rotatable disc (not shown) by means of bolts passed through openings in

circumferentially spaced lugs 2 in the drag-taking member 1.

The stationary drag-taking member 1 is of one-piece construction, preferably a casting, comprising a circumferentially extending bridge piece 3 which is located in a plane parallel to the plane of the disc and which is of substantial axial length. A pair of axially extending circumferentially spaced guiding members 4 integral with the bridge piece 3 and extending upwardly and outwardly with respect to the bridge piece 3, are constituted by spaced arms.

The oppositely facing side faces of the guiding members 4 are of stepped outline and spaced parallel portions 5 at the innermost ends of the side faces constitute guiding surfaces and at their outermost ends lead into spaced parallel guiding surfaces 6 which are spaced apart by a distance greater than the distance between the guiding surfaces 5.

The radially outermost ends of the guiding surfaces 6 lead into outwardly directed aligned shoulders 7.

A caliper 8 of generally U-shaped outline including a pair of opposed limbs straddles the disc and is mounted on the drag-taking member 1 between the guiding members 4.

Radially spaced pairs of circumferentially opposed end faces 9 and 10 of one limb of the caliper are in sliding engagement with the guiding surfaces 5 and 6 respectively in the arms 4.

The radially outermost end of the one limb is formed with opposed outwardly or circumferentially directed abutment surfaces 11 continuous with the end faces 10 and the abutment surfaces 11 are complementary to and co-operate slidably with the shoulders 7 at the outermost ends of the arms 4.

The one limb incorporates brake applying means in the form of an hydraulic piston 12 working in a bore 13 in the one limb.

A first friction pad assembly 14 in the form of a friction pad 15 carried by a rigid backing plate 16 is interposed between the piston 12 and the disc and is guided between the oppositely facing guiding surfaces 5.

A second friction pad assembly (not shown) for engagement with the face of the disc remote from the stationary member 1 is carried by the other limb of the caliper 8.

In the application of the brake, hydraulic fluid under pressure is admitted to the closed end of the bore 13 and advances the piston 12 in the bore 13 to apply the first friction pad assembly 14 to an adjacent face of the disc. Simultaneously the reaction on the caliper 8 causes the caliper to slide relative to the stationary member 1 in the opposite direction to apply the second friction pad

assembly to the opposite face of the disc. In a normal direction of disc rotation the drag on the directly actuated friction pad assembly 14 is taken directly by the trailing arm 4. That is to say the drag is taken by the arm with which any point in the braking surface of the disc last comes into alignment when the disc is rotating in that direction. The drag on the indirectly actuated friction pad assembly is transferred to the arms 4 through the caliper 8. Ideally the drag from the indirectly actuated friction pad assembly will be taken by the guiding surfaces 5 and 6 on the trailing arm 4, since that arm extends over the disc to a point coinciding with the line of action of the force of the drag due to the indirectly actuated friction pad assembly. However, due to tolerance variations, it is possible that the line of action of the force from the drag on the indirectly actuated friction pad assembly will no longer be coincidental with the guiding surfaces 5 and 6. In that case the caliper 8 is subjected to a turning movement which acts on the guiding surfaces 5 and 6 of the trailing arm 4 and reacts on the guiding surfaces 5 and 6 of the leading arm.

Separation and movement of the caliper 8 in a radially outwards direction with respect to the drag-taking member 1 is controlled by the provision of restraining means.

As illustrated in the drawings, aligned recesses 17 are provided in the inner face of the arms 4 of the drag-taking member 1. Each recess 17 is spanned by the stem of an axially extending bolt 18. Each bolt 18 extends inwardly from the outer end of the drag-taking member 1 and the free end of a portion 19 of reduced diameter is received in a recess 20 in a wall defining the inner face of the recess 17. The portion 19 forms a pivotal connection for a lever 21 of the outline of a snail cam. The lever 21 is of double thickness pressed steel construction and is of generally circular outline having an eccentrically arranged opening 22 in which the portion 19 is received. A pair of aligned oppositely directed lugs 23 lying in a plane normal to the plane of a portion of the lever 21 which contains the opening 22 are produced by deforming in opposite directions portions of the lever which lie below a split line 24 extending radially inwardly from the edge of the lever 21 remote from the opening 22 and lying on a line passing through the centre of the opening 22. The faces of the lugs 23 and the edge of the lever at the split line 24 define a pair of arms arranged at right angles to each other for engagement with a radial surface 25 and shoulder 26 formed in the caliper 8. The shoulder 26 is located above and at right angles to the radial surface 25.

At a position spaced above a line con-

maintaining the split 24 and passing through the centre of the opening 22 each lever 21 is formed with a pair of circumferentially spaced openings 27 and 28. Each opening 27 lies on a line passing substantially through the centre of the opening 22 at right angles to the split line 24, and the opening 28 lies substantially in a plane containing the faces of the lugs 23.

One opening 27 or 28 of each lever 21 forms an anchorage for the lower end of a tension spring 29 of which the outer end is connected to an anchorage 30 on the portion of the caliper 8 containing the abutment surface 11.

Depending upon which opening 27 or 28 forms the anchorage for the spring 29, the spring force applied to the caliper 8 acts inwardly in either a circumferential direction or in a radial direction. As illustrated in the drawings one of the levers 21 applies a circumferentially directed force to the caliper 8, and the other lever 21 applies a radially directed force to the caliper. Normally the radial force will be applied to the caliper at the leading end of the brake and the circumferentially directed force will be applied at the trailing end.

The provision of the levers 21 and the tension springs 29 serve to ensure that the abutment surfaces 11 are urged into engagement with the shoulder 7, and that at least the surfaces 5 and 9 and 6 and 10 at one end of the brake are urged into engagement. This has the advantage of taking up clearances to reduce undue noise in the application of the brake and reduce unwanted relative movement between the caliper 8 and the drag-taking member 1 in the normal operative position.

In the event of failure of one of the tension springs 29 the lever 21 to which that spring is connected serves as a stop to prevent free movement of the caliper 8 away from the drag-taking member 1 in a radially outwards direction. This is achieved by means of a wedging action between the arms of the lever and the shoulder 26 and the surface 25 with which they normally engage alternatively when subjected to loading from the tension spring 29.

In the embodiment described above our invention has been described with reference to a brake of the kind in which the stationary drag-taking member is located adjacent to one face of the disc, and hydraulic actuating means, which are located in one limb of the caliper member, are adapted to apply one friction pad assembly to one face of the disc, the other friction pad assembly being applied to the opposite face of the disc by the reaction of the actuating means which cause the caliper member to slide with respect to the stationary member in a direction parallel to the axis of the disc. It is

to be understood, however, that our invention can also be applied to constructions of brake in which the caliper member is provided in each limb with at least one hydraulic actuator, and also to those constructions of brake in which the stationary drag-taking member comprises a U-shaped member straddling the periphery of the disc and provided with a circumferentially extending gap between opposite ends of which the caliper member is guided to slide relative to the disc.

#### WHAT WE CLAIM IS:—

1. A disc brake of the kind set forth for vehicles, in which the resilient means acts on the lever at one of a pair of relatively spaced locations and, when the resilient means acts at one location, one pair or set of complementary surfaces are urged into engagement, another pair or set of complementary surfaces being urged into engagement when the resilient means acts on the other location.

2. A disc brake as claimed in Claim 1, in which complementary circumferentially arranged surfaces on the members are urged into engagement when the resilient means acts on the lever at one location, and complementary radially arranged surfaces on the members are urged into engagement when the resilient means acts on the lever at the other location.

3. A disc brake as claimed in Claim 1 or Claim 2, in which the lever is formed with a pair of abutment surfaces which are disposed at right angles to each other and which intersect at their inner ends, each abutment surface being engageable with a different one of a pair of complementary surfaces on said one member and the said one surface is disposed at right angles to the other surface of that pair with which the other abutment surface is adapted to engage when the resilient means acts on the lever at the other location.

4. A disc brake as claimed in Claim 4, in which the pivotal connection between the lever and the said other member, the abutment surfaces and the pair of complementary surfaces are relatively arranged so that the abutment surfaces define stop means to prevent substantial relative movement between the members in a radial direction, upon failure of the resilient means.

5. A disc brake as claimed in any one of the preceding claims, in which a lever and a resilient means are located at each of the circumferentially spaced opposite ends of the brake and each resilient means acts between a fixed anchorage on the said one member and one of the locations on the corresponding lever at that end of the brake.

6. A disc brake as claimed in Claim 5, 130

in which each lever is pivotally connected to the stationary member by means of an axially extending pivotal connection, and each resilient means comprises a tension spring acting between the fixed anchorage and the corresponding lever.

5 7. A disc brake as claimed in Claim 5, in which one of the abutment surfaces on each lever acts on a complementary surface on the caliper member under the influence of the tension spring acting between that lever and the complementary anchorage on the caliper member.

10 8. A disc brake as claimed in any one of Claims 5 to 7, in which connection between the resilient means and the levers are constructed and arranged such that one lever subjects the caliper member to a circumferentially directed force and the other lever subjects the caliper member to a radially directed force.

15 9. A disc brake as claimed in any one of Claims 3 to 8, in which the or each lever is of generally circular outline having an eccentrically arranged opening co-operating with the pivotal connection, and a pair of aligned oppositely directed lugs lying in a plane normal to the plane of a portion of the lever containing the eccentrically arranged opening define the abutment surface

for engagement with a first surface on the said one member.

10. A disc brake as claimed in Claim 9, in which the locations comprise circumferentially spaced openings in the lever, one of the circumferentially spaced openings lying on a line passing substantially through the centre of the eccentrically arranged opening in a direction normal to the second surface on the lever, and the other circumferentially spaced opening lying in a plane containing the abutment surface of the lugs.

11. A disc brake as claimed in Claim 9 or Claim 10, in which the or each lever is of double thickness pressed steel construction and the lugs are formed by deforming in opposite directions portions of the lever which lie on the side of a split line remote from the eccentrically arranged opening.

12. A disc brake of the kind set forth for vehicles, substantially as described with reference to or as illustrated in the accompanying drawings.

BARKER, BRETTELL & DUNCAN,  
Agents for the Applicants,  
Chartered Patent Agents,  
16, Greenfield Crescent,  
Edgbaston,  
Birmingham, B15 3BA.



